ACOPOS Control Concept and Adjustment TM450

Prerequisites

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1. INTRODUCTION

The high level of performance of a servo drive's controller is crucial for the quality, precision and dynamic capabilities during a process. This means that the control concept as well as the controller settings are decisive factors.

Fig. 1 ACOPOS servo drive and motor

During this training we will first learn the basics before taking a step-bystep look at the control concept of the ACOPOS servo drive.

We will then take some time learning how to determine the optimum control parameters. Some of the components from Motion Components (training module TM410) will help us throughout this training module.

1.1 Objectives

The course participant will become familiar with the structure and the effectiveness of the ACOPOS control concept.

Each course participant will be able to adjust and optimize the control parameters.

Fig. 2 Overview

2. THE BASICS OF CONTROL TECHNOLOGY

To begin our training, we would like to briefly review a few basics before diving into the details of the ACOPOS control concept.

Let's start by asking ourselves the following question:

Why does a servo drive need a controller?

The goal of a servo drive is to reach a position as fast and as accurately as possible using a motor and various mechanics.

Fig. 3 ACOPOS servo drive and motor

The first thing the controller must know is where to send the motor. This is generally referred to as the **set position**. In control technology, a **reference variable** is used to do this.

The controller must receive some information to find out where the motor is presently located. This information is obtained using an encoder. The encoder is a **measuring element**, which provides the controller with the information (actual position of the motor) via **feedback**.

Fig. 4 ACOPOS servo drive, motor and feedback

A **lag error** or **control deviation** occurs when the set position and the actual position do not match.

This lag error is compensated for by the servo drive. This is why it outputs a **manipulated variable**. This manipulated variable affects the **controlled system** (in this case, the motor and the subsequent mechanics) so that the actual position reaches the set position. In our case, the actual position is the **control variable**. This lag error is compensated for by the servo drive. This is why it outproved a
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avatem (in this case, the mortor and the subsequent mechanics) so that

All of this together is called the **closed control loop**.

Fig. 5 ACOPOS servo drive, motor, closed control loop and disturbance variables

There are even more external factors that affect this system. These types of factors are called **disturbance variables**, and also must be compensated for by the controller. A suspended load is one example of a disturbance variable.

Differences between closed and open loop controllers:

Unlike a closed loop controller, an open loop controller does not have feedback. This means we cannot determine when, how or whether the goal has been reached. An open loop controller is generally used for conventional frequency converters.

But now what happens within a closed loop controller?

A closed loop controller can be made up of one or more parts of transfer elements. These transfer elements react differently to input values.

The controllers on the ACOPOS servo drive are generally made up of a **proportion element** (P-element) and an **integral element** (I-element).

A P-element immediately reacts to an input value jump with a proportional output value jump. The size of the jump on the output is determined by a factor. This is labeled as the factor "**kv**".

Fig. 6 Reaction of a P-element

The output value increases continuously in the form of a ramp if the input value of an I-element jumps. The slope of this ramp and the rate at which the output value increases depends on the time. We will label the time as integral action time "**tn**".

Fig. 7 Reaction of an I-element

A **PI controller** contains a P- and an I- section. The output values of both elements are added together. As a result, the PI-controller reacts to a jumping input value as follows.

Fig. 8 Reaction of a PI-controller

The P-section allows the controller to react immediately to a change in the input value. A remaining controller deviation would occur if only one Psection was used alone. The I-section integrates this, thereby compensating for the deviation.

The ACOPOS servo drive is equipped with a high-performance processor. Among other tasks, this processor also calculates the controller algorithms. This is why the term "digital control" is used. The difference between an analog (continuous) controller and a digital (discrete time) controller is that the controller deviations is scanned in a corresponding timeframe (cycle). The duration of this cycle should always be the same $($ = jitter-free), but should also be as short as possible to receive the changes in the controller deviation as fast as possible. jumping input value as follows.

^e the controller

Fig. 3 Reaction of a Pi-controller

Fig. 3 Reaction of a Pi-controller

Fig. 3 Reaction allows the controller to react immediately to a change in high part value.

Fig.

3. CASCADED CONTROLLER CONCEPT

3.1 Overview

The cascaded controller concept is the core of the ACOPOS servo drive. The **position controller**, **speed controller** and **current controller** are cascaded starting with the set value generator. As a result, the manipulated variable of the higher-level controller becomes the reference variable for the lower-level controllers (e.g.: the position controller determines the set speed for the speed controller).

Fig. 9 Cascaded structure

3.2 Set value generator

Fig. 10 Principle

3.2.1 Basis movements

As seen earlier, the reference variable is provided for the position controller by a set value generator. This is done with a cycle time of **400µs**.

The job of this set value generator is to create a movement profile after a command for executing a basis movement. The course of this profile depends mostly on the **basis movement parameters** (target position, acceleration, etc.).

Fig. 11 Automation Studio online help, basis movements

The following figure illustrates this type of profile and the parameter values used:

Fig. 12 Trace display of the position, speed and acceleration (derivative of the speed)

We can see from the chart that jumps occur at each end of the acceleration. These jumps are called **jolts**. These occur when there is a bend in the respective speed curve.

Generally this type of behavior is not desired because the motor must generate a higher torque. Furthermore, a high load is placed on the mechanics and the entire system vibrates.

This is why the set value generator in an ACOPOS servo drive is provided with jolt limitation.

3.2.2 Jolt limitation

As we have already learned, the jolt occurs due to a change that causes a bend in the speed curve. If the speed is slowly increased at the beginning or slowly reduced at the end of the acceleration phase, the rectangular acceleration profile becomes a trapezoid. The following chart illustrates this point for us (the basis movement parameters were set the same as in Fig. 12):

Fig. 13 Trace display of the position, speed and acceleration with active jolt filter

A jolt filter time can be set for the ACOPOS servo drive to generate a movement profile with jolt limitation.

The jolt filter time is the time required for acceleration from zero to the maximum value defined. Jolt limitation uses a linear filter during runtime.

In Fig. 13 we can see that the jolt filter time "**t_jolt**" has been set to 0.03 seconds. The measurement cursors in the lower chart diagram indicate that the rise time of the acceleration is the same as the jolt filter time.

An active jolt limitation extends the time for the set value generation (see Fig. 14). However, in many cases the positioning goal can be reached sooner because the settling time of the system is considerably shortened by the lower jolt load.

Fig. 14 Timing diagram of a movement with (bold line) and without (dotted line) jolt limitation

3.3 Predictive position controller

The reference variable of the position controller is created by the set value generator. The ACOPOS servo drive receives the control variable (current motor position) via the motor's encoder system and a corresponding encoder interface card. The control deviation is determined from these two variables, which results in a new manipulated variable for the lower-level speed controller.

Fig. 16 Block diagram of the position controller

The position controller is implemented as PI controller with anti-windup (manipulated variable limitation) and "predictive" feed forward (input control).

The proportional element with the factor "**kv**" causes an immediate change in the set speed in the event that a lag error occurs. Changes in the set value or disturbance variables can cause these type of control deviations.

The I-element with integral action time "**tn**" is used to compensate for stationary disturbance quantities (e.g. suspended loads).

The manipulated variable limitation is implemented using the parameters "**p_max**" and "**i_max**". These values limit the maximum effect of the Psection and the I-section.

The feed forward is the predictive element of the position controller.

Fig. 17 Prediction, feed forward

A feed forward speed ("v_feed") results from a differentiation of the set position (s_set). The PI-controller establishes a correction speed ("v_corr") from the lag error (∆s). These two speeds are added together to produce the set speed (v_set) for the subsequent speed controller. The feed forward is then predictive if the set position is sent to the PI-controller with a delay (t_total). The set value should be delayed as long as the delay time of the controlled system. This is why the set speed $(t$ total $- t$ predict) is introduced to the speed controller first. If the corresponding set position of the PI-controller is then accepted, the control deviation is then smaller as a result of the already fed set speed value. This presents a load on the PIcontroller because it still has to compensate for the remaining deviation. Without speed input control, the PI-controller would have to take care of the set speed by itself. This improves the reference behavior and the dynamic properties of the drive. value or disturbance variables can cause these type of control deviation

The Lelement with integral action time "the" is used to compensate for

The manipulated variable limitation is implemented using the parameter

"p_m

Calculation of "v_corr" which results from the lag error "∆s":

Fig. 18 Automation Studio online help, calculation of "v_corr"

Finally "**v** corr = **v** $p + v$ i" can be calculated.

The set speed is converted from the configurable measurement system (unit/sec) to the physical motor encoder system (rev./sec) before being passed onto the speed controller.

Lag error monitoring is also performed in the position controller. An E-Stop is executed if the control deviation exceeds a configurable threshold value.

3.4 Speed controller

3.4.1 Speed controller – General function

The speed controller's job is to determine the difference between the manipulated variable of the higher-level position controller and the measured speed. A manipulated variable for the lower-level current controller is then generated using a **PI-controller** with anti-windup.

Fig. 19 Block diagram of the speed controller

The set speed value from the position controller is first fed through an interpolator. This is necessary because the position controller runs at a cycle time of 400µs, whereas the speed controller runs at **200µs**. The value is then limited using the maximum motor speed.

The actual speed is determined by differentiating the encoder position. The value is then sent through the **speed filter**. The value is converted from "incr./sec" to the unit "rev./sec" before it can be subtracted from the set speed. cycle time of 400₀₄₈, whereas the speed controller runs at 200*µs*. The variably cycle time of 400₀₄₈, whereas the speed controller runs at 200*µs*. The variable is then limited using the maximum motor speed.
The actua

We will take a more detailed look at the functionality of the speed filter a little later.

The P-element with the factor "**kv**" allows the controller to react immediately to any control deviation. This makes this element decisive for the dynamics of the speed controller.

The I-element with integral action time "**tn**" is used to compensate for stationary disturbance variables (e.g. load torque).

The output of the PI-controller can be filtered using a notch filter.

The function of the notch filter will be explained in a later section.

Before the set current can be provided as the reference variable for the current controller, the value is limited using a torque limiter. This torque limitation also determines the anti windup limits for the speed controller Ielement.

3.4.2 Torque limiter

The torque limiter is mostly used to protect the motor and the ACOPOS servo drive from the following risks:

- The ACOPOS servo drive cannot output more current than the motor can handle (motor peak current).
- The motor's stator current cannot exceed the ACOPOS peak current.

By default, the torque limiter is initialized using the smaller of the following two values:

- Motor peak current
- ACOPOS peak current

3.4.3 Signal filter

Signal filters can be used on ACOPOS servo drives to separate high frequency disturbances (e.g. signal noise) from a desired signal and to suppress a resonance frequency.

Disturbance in the encoder signal can be caused by one or more of the following possibilities:

- Coupling of disturbances on the communication path (encoder cable).
- Quantization noise when converting the analog signal to a digital form. This mostly occurs when evaluating a resolver signal with low resolution.

With warp-resistant drive mechanics (e.g. direct load coupling to the motor shaft using fastening devices) the mechanical system can be subject to oscillations due to the closed control loop ("two-mass oscillation"). These types of systems generally have a resonance frequency in the range from 700 to 1500Hz. This is further dependent on the following factors:

- Rigidity of the mechanical system
- Mass inertia of the mechanical system
- Physical layout of the system.

Speed filter

As described in section 3.4, the actual speed is filtered, before being processed in the speed controller. This filter is a "speed filter" and functions like a low pass. High-frequency disturbances can be filtered out from the speed signal using this low pass behavior. This allows us to achieve higher controller quality. Signal filters can be used on ACOPOS servo drives to separate high

signal filters can be used on ACOPOS servo drives to separate high

suppress a resonance frequency.

Disturbance in the encoder signal can be caused by o

Caution:

Parts of the desired signal will also be filtered out if the limit frequency of the low pass filter is set too low!

Notch filter

The frequency range of the set current, which causes the mechanical system to oscillate, can be filtered for the current controller.

Caution:

The notch filter should only be used on mechanics with a rigid coupling (e.g. direct drive). This filter should not be used for connections such as belts or gears!

Furthermore, it can only be used if the existing moments of inertia are always constant!

The resonance frequency of the system could shift as a result of mechanical wear. This means that over time the defined filter can lose its effectiveness.

The notch filter is only effective when the resonance frequency is in the range from 700 to 1500Hz.

The filter has the highest amount of damping at the "notch frequency" entered (= resonance frequency of the mechanical system). There is a range (band width) around this notch frequency in which the damping is lower than 3dB. The smaller the band width is set, the stronger the damping is in the notch frequency. The frequency range of the set current, which causes the mechanical
system to oscillate, can be filtered for the current controller.
Caution:
The notch filter should only be used on mechanics with a rigid couplin
(e.g. d

The notch filter can be used (once all of the requirements for use have been met) to increase the controller gain factors, without causing the entire system to become unstable.

3.5 Current controller

The current controller is made up of PI-controllers (like the position and speed controllers). The corresponding parameters are automatically determined by the servo drive using the motor parameters and the specific ACOPOS parameters. This means that we do not have to spend time in a later section going over how to set these values.

The current controller uses its manipulated variable to control the IGBTs (Insulated Gate Bipolar Transistor). These then output a pulse width modulated (PWM) current signal to the motor.

The current controller function at different cycle times depending on the PWM switching frequency:

4. THEORETICALLY DETERMINING THE CONTROL PARAMETERS

In the previous sections we learned how the ACOPOS servo drive controllers are structured and inter-related. Now we want to find the corresponding values for the control parameters. These values can be calculated or determined empirically if some of the requirements have not been met.

We can use the following formulas to determine good output values for the control parameters in the event that we don't already know the system's **total moment of inertia** and if the **load is fixed to the motor**. In most cases, we will achieve even better controller behavior by making fine adjustments to the parameter values manually. is sections we identical now the ACUPOS servol direct set of structured and inter-related. Now we want to find the y values for the control parameters. These values can be determined empirically if some of the requirement

4.1 Speed controller

Replacement time constant ${\mathsf T}_{\mathsf I}$ of the current control loop:

 $\overline{}$ J \backslash $\overline{}$ l ſ × $= 2 \times | 0.000075 +$ $T_I = 2 \times \left(0.000075 + \frac{1}{2 \times SwitchingFrequency}\right)$ $2 \times \left(0.000075 + \frac{1}{2(1.000075)^2}\right)$

Summation of the individual time constants to a replacement time constant T_{σ} v:

 T_{σ} $_{v} = T_{I} + T_{dead}$ $_{v} + T_{filter}$

 T_{dead} _v = 0.000175 s (encoder interface dead time, speed determination und scanning)

 T_{filter} ("t_filter" parameter) = Filter time constant of the speed filter

Proportional gain of the speed controller:

Proportional gain of the s

\n
$$
kv = \frac{J \times \sqrt{2} \times \pi}{T_{\sigma_{\text{av}}} \times k_{\text{r}}}
$$
\nJ = total moment of inertia

\n
$$
k_{\text{t}} = \text{Torque constant of the}
$$
\nIntegral action time of the

\n
$$
tn = 4 \times T_{\sigma_{\text{av}}}
$$

J = total moment of inertia $(J_{\text{motor}} + J_{\text{brake}} + J_{\text{load}})$

 ${\sf k}_{\scriptscriptstyle \rm t}$ = Torque constant of the motor being used [Nm/A]

Integral action time of the speed controller:

 $tn = 4 \times T_{\sigma}$

4.2 Position controller

Summation of the individual time constants to a replacement time constant T_{σ_p} : Summation of the individual time constants to a replacement time cons
 $T_{\sigma_{\alpha}p}$:
 T_{α} , $T_{\text{linear}} + 4 \times T_{\alpha_{\alpha}} + T_{\text{dual}_{\alpha}q}$
 $T_{\text{inner}} = \text{Dead time resulting from interpolator (0.0001s)}$
 $4 \times T_{\sigma_{\alpha}y} = \text{Replacement time constant of the speed control loop}$
 $4 \times T_{\sigma_{\alpha}y} = \text{Dead time resulting from scanning (0.0$

 $T_{\sigma_{p}} = T_{\text{int }erpol} + 4 \times T_{\sigma_{p}} + T_{\text{dead}}$

 T_{interpol} = Dead time resulting from interpolator (0.0001s)

 $4 \times T_{\sigma_{-}v}$ = Replacement time constant of the speed control loop

 $T_{dead,p}$ = Dead time resulting from scanning (0.0002s)

Proportional gain of the position controller:

$$
kv = \frac{1}{2 \times T_{\sigma_{-p}}}
$$

Integral action time of the position controller:

 $tn = 4 \times T_{\sigma_{p}}$

Example:

Configuration of the 8MSA2S.E0 motor with ACOPOS 8V1010.00-2 without load:

 $k_t = 0.46$ Nm/A

 $J = 0.06$ kgcm²

Switching frequency = 10000 Hz

Speed controller:

Configuration of the 8MSA2S.E0 motor with ACOPOS 8V1010.00-2

\nwithout load:

\n
$$
k_{t} = 0.46 \text{ Nm/A}
$$
\n
$$
J = 0.06 \text{ kgcm}^2
$$
\nSwitching frequency = 10000 Hz

\nSpeed controller:

\n
$$
T_{t} = 2 \times \left(0.000075 + \frac{1}{2 \times SwitchingFrequency}\right) = 2 \times \left(0.000075 + \frac{1}{2 \times 10000}\right) = 0.00025 \text{ sec}
$$
\n
$$
T_{\sigma_{-v}} = T_{t} + T_{dead_{-v}} + T_{filter} = 0.00025 + 0.000175 + 0 = 0.000425 \text{ sec}
$$
\n
$$
kv = \frac{J \times \sqrt{2} \times \pi}{T_{\sigma_{-v}} \times k_{t}} = \frac{0.000006 \times \sqrt{2} \times \pi}{0.000425 \times 0.46} = 0.136 \text{ As/Re} v.
$$
\n
$$
tn = 4 \times T_{\sigma_{-v}} = 4 \times 0.000425 = 0.0017 \text{ sec}
$$

$$
T_{\sigma_{-v}} = T_I + T_{dead_{-v}} + T_{filter} = 0.00025 + 0.000175 + 0 = 0.000425 \text{ sec}
$$

$$
kv = \frac{J \times \sqrt{2} \times \pi}{T_{\sigma_{v}} \times k_{t}} = \frac{0.000006 \times \sqrt{2} \times \pi}{0.000425 \times 0.46} = 0.136 \text{ As/Re} v.
$$

 $tn = 4 \times T_{\sigma}$ = $4 \times 0.000425 = 0.0017 \text{ sec}$

Position controller:

 $T_{\sigma_{p}} = T_{\text{int } epoch} + 4 \times T_{\sigma_{p}} + T_{\text{dead}} = 0.0001 + 4 \times 0.000425 + 0.0002 = 0.002$ sec $T_{\sigma_P p}$ *kv* $2 \times T_{\sigma}$ 1 $\times T_{\sigma}$ $=\frac{1}{2 \cdot \pi}$ = sec $\frac{250}{ }$ $\frac{1}{ }$ 2×0.002 $\frac{1}{2}$ = × $tn = 4 \times T_{\sigma_{p}} = 4 \times 0.002 = 0.008 \text{ sec}$ Position controller:
 $T_{\sigma_{p}} = T_{\text{interpol}} + 4 \times T_{\sigma_{p}} + T_{\text{dec}}$
 $kv = \frac{1}{2 \times T_{\sigma_{p}}} = \frac{1}{2 \times 0.002} =$

5. PROCEDURE FOR SETTING THE CONTROLLER

In this section we will learn about a possibility for determining control parameters, which has been proven time and time again in the field. This will allow you to check and make fine adjustments to values that have already been calculated. If this is not possible, suitable values can be determined empirically. In this case, the values specified in the corresponding notes can be used as start values.

5.1 General information

The control parameters only really have to be determined when the mechanics are already put together. When dealing with a machine where the axis is loaded with different masses, the parameters must be tested both without load and with the highest load. The parameters should also be tested at different speeds and accelerations.

These tests could result in the need for a compromise.

Globally valid control parameters cannot be used because all mechanics have different features.

To determine parameters from cascaded controllers, it is best to start from the bottom (last) controller and work up. In our case, this means that we would start by setting the speed controller first followed by the position controller. The current controller is automatically configured by the ACOPOS servo drive.

Fig. 20 Order for setting the controllers

It is usually the goal to set the controller as "tough" as possible. A controller can be considered "tough" when a disturbance variable is compensated for as quickly and perfectly as possible.

Example:

We want to manually rotate a flywheel mass which is mounted to the motor shaft.

- The controllers are set "soft" if the flywheel mass can be easily rotated.
- The controllers are set "hard" if the flywheel mass is difficult to rotate or cannot be rotated at all.

However, sometimes it is not the goal to set the controller as tough as possible. This is because a tough controller can cause quick heating, a higher load on the mechanics and therefore more wear. This is the reason why a compromise must often be found when determining the parameters.

We will be using the Motion Components test window to help us set the control parameters. This allows us to change and initialize the control parameters online. This also makes it possible to start positioning movements using any basis movement parameters. We can also configure, start and evaluate the trace in the Motion Components test window. compensated for as quickly and perfectly as possible.

Example:

We want to manually rotate a flywheel mass which is mounted to the

motor shaft.

• The controllers are set "soft" if the flywheel mass can be

easily rotate

The behavior of the controller during a typical movement can be closely determined by selecting suitable parameters for tracing. Therefore, a basis movement (e.g. relative or absolute movement) is usually started and the respective parameters are recorded.

Furthermore, the control parameters should be selected so that oscillation of the recorded values is minimal.

Examples:

Fig. 21 Strong oscillations

Fig. 22 Almost no oscillations

The user must then check to make sure that all of the requirements have been met (e.g. lag error within the tolerance) once all parameters have been set.

If this is the case, make sure that there are reserves for the gain factor because the system can behave differently due to mechanical wear. Therefore, a corresponding reserve (approx 1/3) should be taken from the determined values.

5.2 Speed controller

The following three parameters can be set for the speed controller. These are located in a subgroup of the control parameters:

Fig. 23 Automation Studio online help system – Controller

Note:

The following parameters can be configured for the trace when setting the speed controller:

- Set speed: ACP10PAR_SCTRL_SPEED_REF (ID 250) [1/sec]
- Actual speed: ACP10PAR_SCTRL_SPEED_ACT (ID 251) [1/sec]
- Current controller: Set stator current of quadrature component $[A]$

The scan rate should be set as low as possible (approx. 0.2 to 4msec). A trigger event can be used to start the trace as accurately as possible (further information can be found in the Automation Studio Online help).

The "kv" and "tn" parameters of the position controller should be initialized with the value "0" so that only the speed controller is active

Note:

The value from "current controller: set stator current of the quadrature component" is the torque generating component of the set current.

The peak value of the current is displayed in the Motion Components trace. This value must be divided by the factor $\sqrt{2}$ (≈1.414) to compare it with the specifications of the motor parameters and the ACOPOS™ servo drive parameters.

5.2.1 Proportional gain "kv"

The most important parameter of the three speed controller parameters is the gain factor "kv". This parameter significantly determines the dynamic properties of this controller. The goal is to set the value as large as possible without causing the system to oscillate.

Note:

1/5 to 1/10 of the nominal motor current (In) can be used as starting value for this factor.

5.2.2 Integral action time "tn"

For most applications it is not necessary to use an integral action time in the speed controller ("tn" = 0s). However, this time value should not be set too low if an application does require an I-section (e.g. to compensate for load-side disturbances, poor (soft) load coupling, high speed precision). Otherwise, the tendency for oscillation is increased in the speed controller.

Note:

100msec (0.1sec) can be used as starting value for the integral action time if you want to use the I-section.The value can then be gradually reduced.

5.2.3 Speed filter

The filter time constant "t filter" for the speed filter is the last parameter in the speed control parameters. It can be used to set the limit frequency with the unit [sec] (e.g. 1kHz equals 0.001sec).

Improvement to the controller behavior using the speed filter can only be achieved in systems with a high mass moment of inertia and encoder systems with a low resolution (e.g. resolver). Whereas if encoder systems with a high resolution are used, the speed filter generally cannot create any improvements in the controller behavior. The filter time constant "t₁fftler" for the speed filter is the last parameter
the speed control parameters. It can be used to set the limit frequency v
the speed control parameters. It can be used to set the limit freq

Note:

You can start with a value of 0.8msec (0.0008sec). This value can be then be increased in small steps until the controller behavior (less oscillation) has improved. Generally, the usable values are in the range from 0.8msec to 2msec.

5.2.4 Notch filter

Note:

As we already know, the notch filter also works in the speed controller. However, there is no entry for this in the axis structure because it is not used too often. Therefore, it must be directly configured using parameter IDs:

Speed controller – notch filter: Frequency: ACP10PAR_FILTER_F0 (ID 226)

Speed controller – notch filter: Bandwidth: ACP10PAR_FILTER_B0 (ID227)

The resonance frequency of the system can be determined using the following steps:

- Initialize the position controller parameters with the value "0", deactivate the speed filter (t_filter = 0) and deactivate the notch filter (set both parameters to "0").
- Switch on the controller
- The proportional gain of the speed controller is increased until the mechanics oscillate.
- Record the actual speed using the trace (scan rate $= 200 \mu s$). Switch the controller off again when the trace has finished.
- Analyze the frequency spectrum in the trace using FFT (Fast Fourier Transformation).
- Set the most notable frequency occurring in the trace as notch frequency for the notch filter.
- Enter a minimum band width (e.g. 25Hz).
- Switch the controller on again and determine a critical proportional gain for the speed controller.
- If the critical value has been increased, determine if the behavior has further improved due to the variation of the band width and the notch frequency. Otherwise, determine a characteristic natural frequency again. • Initialize the position controller parameters with the value "0",
deactivate the speed filter (t filter = 0) and deactivate the notch filter
sched the speed controller
The proportional gain of the speed controller is i
	- If further improvement is no longer possible, the determined values can be entered in an ACOPOS parameter table for example.

Note:

The notch filter cannot be used if a distinct resonance frequency cannot be determined.

Fig. 24 FFT analysis.

Exercise:

Project: Name

Hardware: Test rack, flywheel mass mounted to the motor shaft.

Specifications:

The basis movement parameters to be used are already defined in the NC INIT parameter module. Project: Name

Hardware: Test rack, flywheel mass mounted to the motor shaft.

Specifications:

The basis movement parameters to be used are already defined in the

NC INIT parameter module.

Positioning path "s" = \pm 5

Positioning path " $s'' = \pm 50000$ Units

A stationary manipulated variable is simulated in the example project using an additive set torque.

Parameters for the trace:

Task:

Use the flow chart (Fig. 32) to set the speed controller in steps.

- What happens in this case if the value "0" is entered for the position control parameter "kv"?
- Can the notch filter be used and defined?
- Can the speed filter be used and defined?
- Would you define an integral action time "tn" for the speed controller?

5.3 Position controller

The parameters for setting the position controller are located in a subgroup of the control parameters:

Fig. 25 Automation Studio online help system – Controller

Note:

The following parameters can be configured for the trace when setting the position controller:

- Position controller: Actual speed [Units/sec]
- Position controller: Lag error [Units]
- Current controller: Set stator current of quadrature component $[A]$

The scan rate should be set as low as possible in this case as well.

5.3.1 Proportional gain "kv"

The value from the "kv" factor should also be set as large as possible for the position controller without causing the controller to oscillate.

5.3.2 Integral action time "tn"

The similar conditions also apply for this value as for the integral action time of the speed controller. For some applications it is sufficient to just use one true P-controller. An I-section must be used if an axis has to compensate for a stationary manipulated variable (e.g. hanging load). Furthermore, it may be necessary to use the I-section of the position controller if the speed controller was only able to be set soft.

The integral action time does not have to be set for the position controller if already set for the speed controller.

Note:

You can also use a starting value of 100msec if necessary in this case.

An I-section in the position controller causes an overshoot when the target position is reached and therefore should only be used in special cases.

5.3.3 Total delay time "t_total"

In a single-axis application, this parameter should be initialized with the same value as the prediction time.

The delay via the network can be compensated using the "t total" in multiaxis applications.

The potential value range is also as large as in the parameter "t predict".

5.3.4 Prediction time "t_predict"

This parameter is required for the feed forward. A value "t predict = $0s$ " disables the offset.

The prediction time makes it possible to compensate for the lag error during the acceleration and deceleration phase to nearly "0". This parameter is generally set after correct values have been determined for "kv" and "tn". In a single-axis application, this parameter should be initialized with the
same value as the prediction time.
The delay via the network can be compensated using the "Lotal" in mu
six applications.
The potential value ran

Note:

The potential value range of the "t_predict" parameter is 0.0 to 0.06 seconds.

Generally, a large prediction time is not really necessary if the speed controller could be set hard.

Definition rule:

0.0002 $(kv_{\text{SpeedController}} \times k_t)$ $\frac{4 \times \pi \times J}{4 \times \pi \times J}$ + × $=\frac{4 \times \pi \times}{4 \times 10^{11}}$ $kv_{SpeedController} \times k_{t}$ t *_ predict* = $\frac{4 \times \pi \times J}{4 \times \pi \times J}$

 $J =$ Moment of inertia on the motor [kgm²] $kv_{speed controller}$ = Proportional gain of the speed controller [As/Rev.] ${\sf k}_{\scriptscriptstyle \rm t}$ = Torque constant of the motor being used [Nm/A]

Note:

The position controller runs at a cycle time of **400µs**. This is why the values for "t_predict" and "t_total" should always be selected in a way so that they are equal to a multiple of 0.0004 seconds. Otherwise, they are rounded off by the ACOPOS™ servo drive.

The following charts display the lag error for different prediction time values.

Corresponding speed profile:

Fig. 26 Corresponding speed curve for the following lag error curves

Fig. 27 Prediction time set too low

"t_predict" set too high:

Fig. 28 Prediction time set too high

"t_predict" set correctly:

5.3.5 Maximum proportional action "p_max"

The influence of the proportional gain can be limited using the "p_max" parameter. This can be done to prevent manipulated variables that are too large.

Note:

The value for these parameters can be calculated using the following formulas:

$$
p_{max} = \frac{I_{max} \times 2}{kv_{speedController}} \times UnitFactor
$$

 I_{max} = Motor peak current [A] $kv_{speed controller}$ = Proportional gain of the speed controller [As/Rev.] Unit factor $=$ Unit scaling [units/rev.]

5.3.6 Maximum integral action "i_max"

The maximum influence of the integral section can be limited using the "i_max" parameter. This can be done to prevent a "windup".

Note:

The value for these parameters can be calculated using the following formula to achieve a required holding torque:

$$
i_{\text{max}} = \frac{\frac{M}{k_t} \times 1.1}{kv_{\text{SpeedController}}} \times UnitFactor
$$

 $M =$ Required holding torque [Nm] ${\sf k}_{\rm t}$ = Torque constant of the motor being used [Nm/A] $kv_{speed controller}$ = Proportional gain of the speed controller [As/Rev.] Unit factor = Unit scaling [units/rev.] The muscline of the proportional gain of the semester and the primarism of the primarity of the semester. This can be done to prevent manipulated variables that are the parameter. This can be done to prevent manipulated v

Exercise:

Same project and hardware as before.

Specifications:

The basis movement parameters to be used are already defined in the NC INIT parameter module.

Positioning path " s " = \pm 50000 Units

A stationary manipulated variable is simulated in the example project using an additive set torque.

Parameters for the trace:

Max. trace duration: 2 seconds Scan rate: 0.0008 seconds

Task:

Use the flow chart (Fig. 32) to set the position controller in increments. This setting will be based on the values determined for the speed controller in the previous exercise. Same project and hardware as before.

Specifications:

The basis movement parameters to be used are already defined in the

NC INIT parameter module.

Positioning path "s" = \pm 50000 Units

A stationary manipulated vari

• What happens if you are not using an integral action time "tn" for the position controller?

5.4 Limit parameter

When the control parameters are optimized, two parameters must still be set for the limit values.

5.4.1 Jolt filter time "t_jolt"

The value for the parameter can be determined by recording the lag errors during a positioning movement without jolt filter time. At the end of this movement, it will become evident that the system must first "settle down". After being determined from the trace, the settling time (time until the oscillations level out) can now be used as jolt filter time "t_jolt".

Fig. 30 Determining the jolt time

5.4.2 Lag error stop limit "ds_stop"

Fig. 31 Automation Studio online help system – Limit values

A warning is output if the current lag error value set for the "ds_warning" parameter is exceeded. An emergency stop is executed if the value of the "ds stop" parameter is also exceeded.

Note:

A controlled e-stop ramp is generated when an e-stop occurs. The set speed is decelerated to "0" using the current initialized limit values.The controller is then switched off.

Note:

The value for "ds_stop" can be determined using the following calculation:

 $\frac{I_{\text{max}}}{I}$ \times 2 \times UnitFactor k V $_{\textit{SpeedController}}$ \times k V $_{\textit{PositionController}}$

 I_{max} = Motor peak current [A] $kv_{speed controller}$ = Proportional gain of the speed controller [As/rev.] kv_{position controller} = Proportional gain of the position controller [1/sec] Unit factor $=$ Unit scaling [units/rev.]

Task:

Same project and hardware as before.

Specifications:

The basis movement parameters to be used are already defined in the NC INIT parameter module.

Positioning path " $s'' = \pm 50000$ Units

A stationary manipulated variable is simulated in the example project using an additive set torque. Same project and hardware as before.

Specifications:

Specifications:

The basis movement parameters to be used are already defined in the

NC INIT parameter module.

Positioning path "s" = \pm 50000 Units

A stationary

Parameters for the trace:

Max. trace duration: 2 seconds

Scan rate: 0.0008 seconds

Task:

Set the limit value correctly and check the result.

5.5 Overview

Fig. 32 Overview of the process for setting the control parameters

6. SAVING THE CONTROLLER SETTINGS

The data just determined must be saved to the controller so that it is available to the ACOPOS servo drive each time the machine is started. All parameters available in the axis structure can be saved in an NC INIT parameter module. The remaining values can be entered in a parameter table.

6.1 NC INIT parameter module

The following options are available for saving the values in the NC INIT parameter module.

• Entering the values directly in the corresponding module.

Fig. 33 NC INIT parameter module

• Saving the determined values immediately in the NC test window.

Once everything has been saved, the project should be compiled and the current version should be transferred to the controller.

6.2 ACOPOS parameter table

The parameters and values (e.g. for the notch filter) can be entered in an ACOPOS parameter table.

Fig. 34 ACOPOS parameter table

7. SUMMARY

By understanding how the parameters that we will be using work, we are now able to determine correct values.

The cascaded control concept allows us to optimize the control parameters and filter parameters step-by-step.

The parameters determined are usually different. However, the procedure for obtaining these values is always the same.

8. APPENDIX

Speed controller:

= × ×= + × ×= + 2 1 000075.02 2 1 000075.02 *SwitchingFrequency* for reprint

 $T_{\sigma_{\perp} v} = T_I + T_{dead_{\perp} v} + T_{filter} = + 0.000175 +$

$$
kv = \frac{J \times \sqrt{2} \times \pi}{T_{\sigma_{-v}} \times k_t} = \frac{\sqrt{2} \times \pi}{\sqrt{2} \times \pi} =
$$

$$
tn = 4 \times T_{\sigma_{-v}} = 4 \times \boxed{\qquad} = \boxed{\qquad}
$$

Position controller:

$$
T_{\sigma_{-p}} = T_{\text{interpol}} + 4 \times T_{\sigma_{-v}} + T_{\text{dead}_{-p}} = 0.0001 + 4 \times \left[+0.0002 = \right]
$$

Position controller:
\n
$$
T_{\sigma_{-p}} = T_{\text{interpol}} + 4 \times T_{\sigma_{-v}} + T_{dead_{-p}} = 0.0001 + 4
$$
\n
$$
kv = \frac{1}{2 \times T_{\sigma_{-p}}} = \frac{1}{2 \times 2}
$$

$$
tn = 4 \times T_{\sigma_{p}} = 4 \times \boxed{\qquad} = \boxed{\qquad}
$$

Notes

Overview of training modules

- TM210 The Basics of Automation Studio **TM600** The Basics of Visualization
TM211 Automation Studio Online Communication TM610 The Basics of ASiV TM211 – Automation Studio Online Communication
TM213 – Automation Runtime TM213 – Automation Runtime **TM630** – Visualization Programming Guide
TM220 – The Service Technician on the Job **TM640 – ASiV Alarm System.** Trend and Di TM640 – ASiV Alarm System, Trend and Diagnostic
TM670 – ASiV Advanced TM223 – Automation Studio Diagnostics TM230 – Structured Software Generation TM240 – Ladder Diagram (LAD)

TM241 – Function Block Diagram (FBD) TM710 – PVI Communication TM241 – Function Block Diagram (FBD)
TM246 – Structured Text (ST) TM711 – PVI DLL Programming TM250 – Memory Management and Data Storage TM712 – PVIServices TM261 – Closed Loop Control with LOOPCONR TM400 – The Basics of Motion Control TM800 – APROL System Concept

TM410 – The Basics of ASiM TM810 – APROL Setup, Configura TM410 – The Basics of ASiM **TM810 – APROL Setup, Configuration and Recovery**
TM440 – ASiM Basic Functions **TM811 – APROL Runtime System** TM440 – ASiM Basic Functions
TM441 – ASiM Multi-Axis Functions TM812 – APROL Operator Managraphy TM812 – APROL Operator Management TM445 – ACOPOS ACP10 Software TM813 – APROL XML Queries and Audit Trail
TM446 – ACOPOS Smart Process Technology TM830 – APROL Project Engineering TM446 – ACOPOS Smart Process Technology TM450 – ACOPOS Control Concept and Adjustment TM840 – APROL Parameter Management and Recipes
TM460 – Starting up Motors TM850 – APROL Controller Configuration and INA TM460 – Starting up Motors

TM850 – APROL Controller Configuration and INA

TM860 – APROL Library Engineering

TM860 – APROL Library Engineering TM860 – APROL Library Engineering TM865 – APROL Library Guide Book TM500 – The Basics of Integrated Safety Technology TM510 – ASIST SafeDESIGNER TM890 – The Basics of LINUX iew of training modules

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